

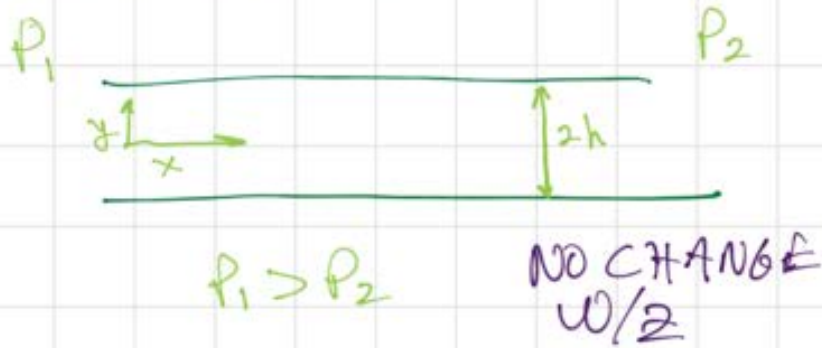
CBE 30357

9/14/17

TOPICS FOR TODAY

- 1) REVIEW OF PRESSURE DRIVEN FLOW
- 2) REVIEW OF GRAVITY DRIVEN FLOW
- 3) FLOWS IN A CYLINDRICAL GEOMETRY
 - HYDROSTATICS
- 4) BOUNDARY CONDITIONS
- 5) SOME LIMITS...

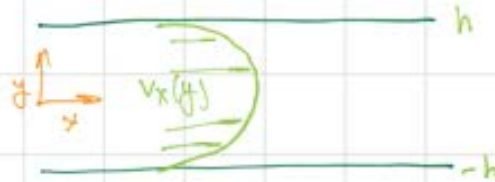
PRESSURE DRIVEN FLOW



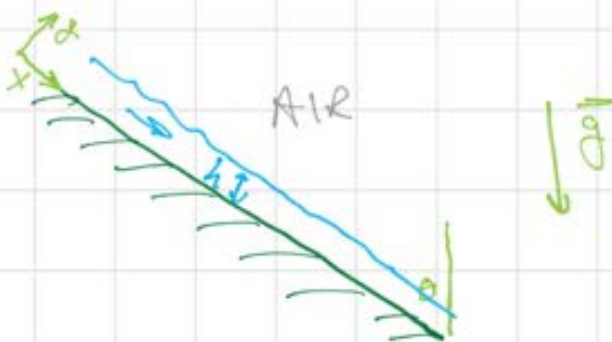
$$0 = -\frac{\partial P}{\partial x} + \mu \frac{\partial^2 v_x}{\partial y^2}$$

SOLUTION FOR v_x IS QUADRATIC IN y

$$v_x(y) = -\frac{1}{\mu} \frac{\partial P}{\partial x} \frac{h^2}{2} \left(1 - \frac{y^2}{h^2}\right)$$

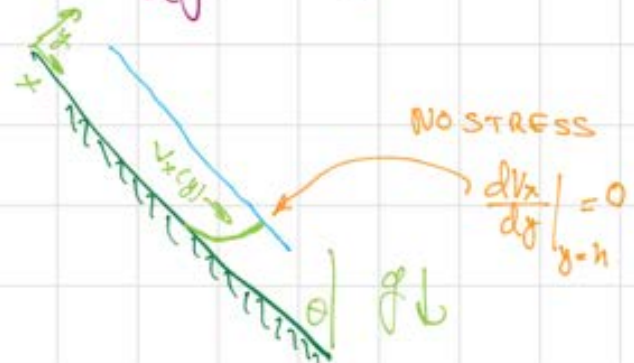


GRAVITY DRIVEN FLOW



$$0 = \mu \frac{d^2 v_x}{dy^2} + \rho g \cos \theta$$

$$v_x(y) = \frac{\rho g \cos \theta}{\mu} \left(\frac{y^2}{2} - yh \right)$$



The mechanism of exercise-induced asthma is . . .

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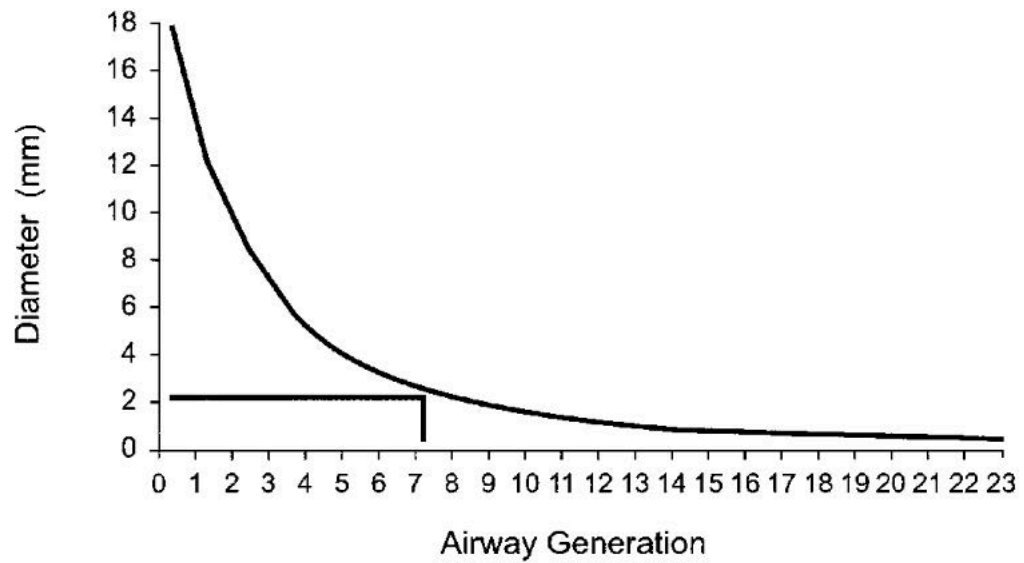
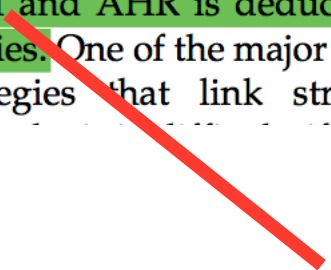


Figure 1. Airway diameters (mm) drawn against airway generations. (Adapted with permission from Ref. 6.)

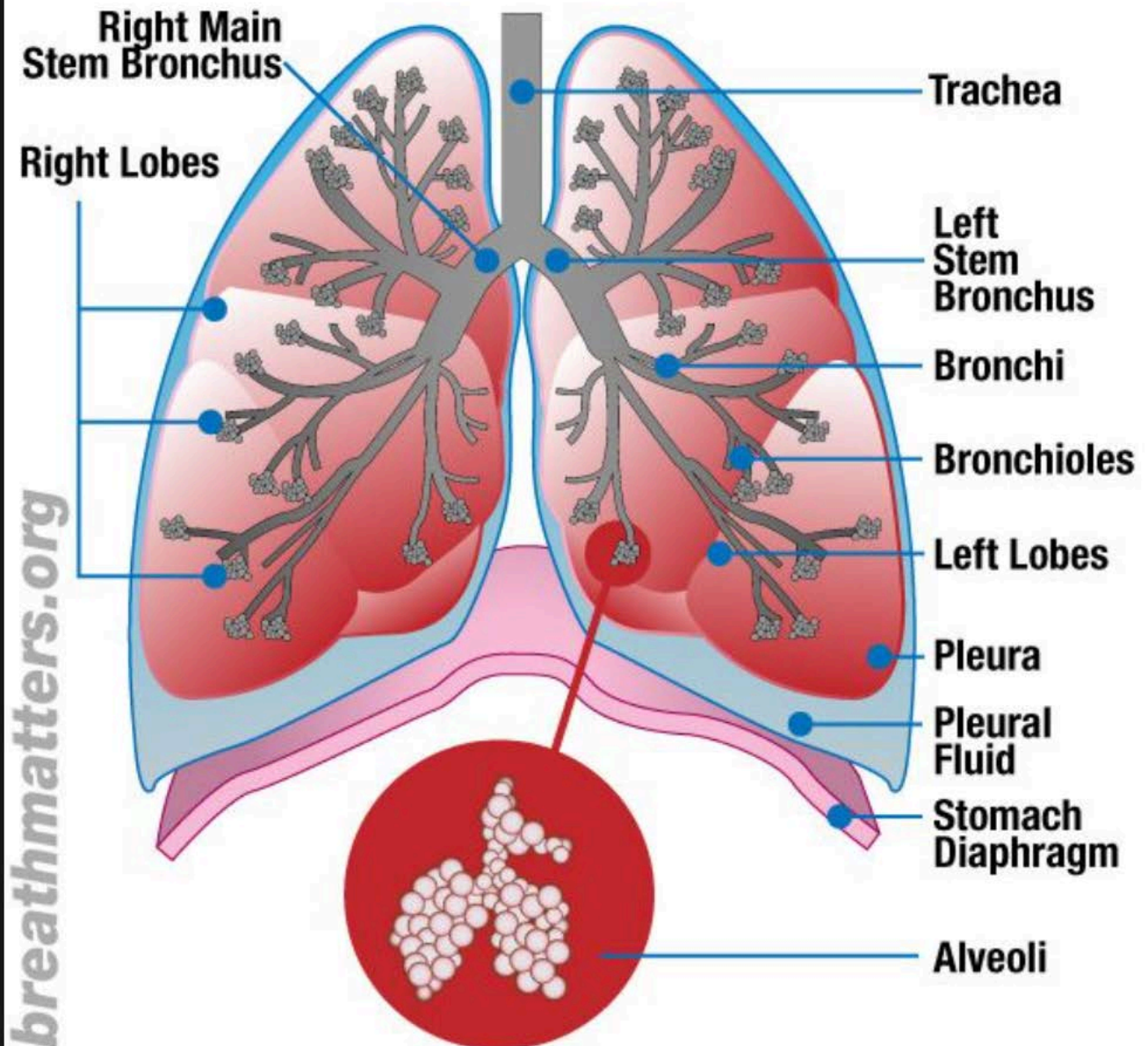
MECHANISMS OF REMODELLING

perhaps best illustrated by irritant-induced asthma. Certain characteristic changes, such as growth of ASM, seem likely to contribute to airways hyperresponsiveness (AHR), but the consequences of other changes, such as subepithelial fibrosis, are not so intuitively obvious. Even the link between growth of ASM and AHR is deduced largely on the basis of modelling studies. One of the major challenges facing the field is to devise strategies that link structure and function directly. For



Airway smooth
muscle (remodeling)

LUNGS



breathmatters.org

Suppose we want to consider the flow in a trachea.

The geometry is a cylinder.

We need the Navier-Stokes equations in cylindrical coordinates.

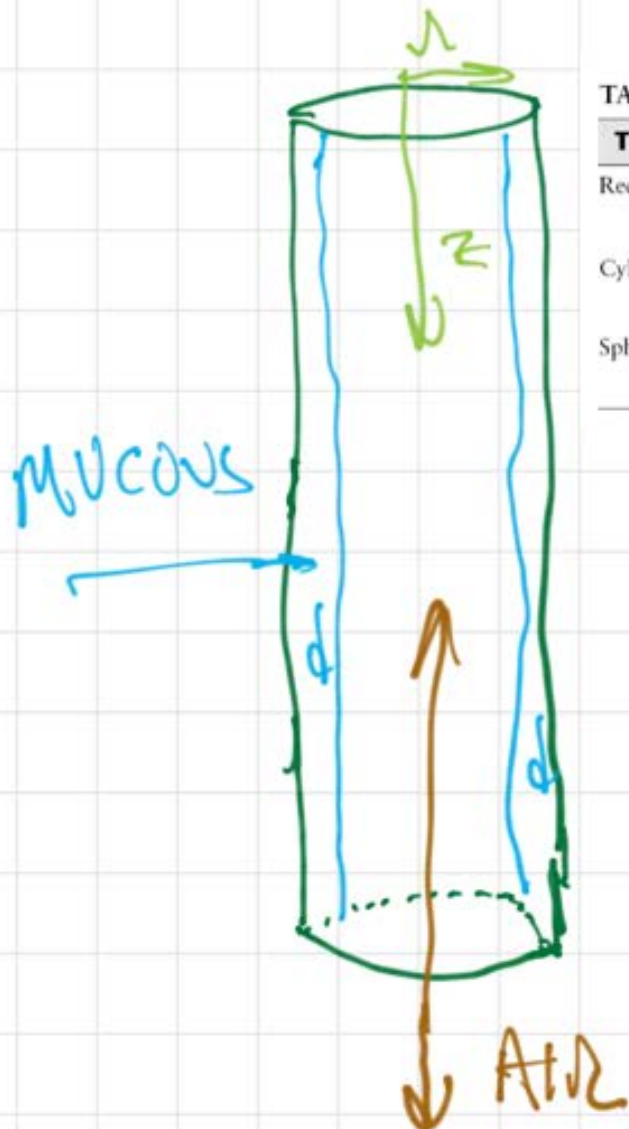


TABLE 3.1

The Conservation of Mass (Continuity Equation)

Rectangular coordinates (x, y, z)	$\frac{\partial \rho}{\partial t} = -\left(\frac{\partial(\rho v_x)}{\partial x} + \frac{\partial(\rho v_y)}{\partial y} + \frac{\partial(\rho v_z)}{\partial z}\right)$
Cylindrical coordinates (r, θ, z)	$\frac{\partial \rho}{\partial t} = -\left(\frac{1}{r} \frac{\partial(\rho r v_r)}{\partial r} + \frac{1}{r} \frac{\partial(\rho v_\theta)}{\partial \theta} + \frac{\partial(\rho v_z)}{\partial z}\right)$
Spherical coordinates (r, θ, ϕ)	$\frac{\partial \rho}{\partial t} = -\left(\frac{1}{r^2} \frac{\partial(\rho r^2 v_r)}{\partial r} + \frac{1}{r \sin \theta} \frac{\partial(\rho v_\theta \sin \theta)}{\partial \theta} + \frac{1}{r \sin \theta} \frac{\partial(\rho v_\phi)}{\partial \phi}\right)$

$$\frac{\partial v_z}{\partial z} = 0$$

$$v_z = v_z(r)$$

ONLY

Cylindrical coordinates

r direction

$$\rho \left(\frac{\partial v_r}{\partial t} + v_r \frac{\partial v_r}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_r}{\partial \theta} - \frac{v_\theta^2}{r} + v_z \frac{\partial v_r}{\partial z} \right) = -\frac{\partial p}{\partial r} + \mu \left[\frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial (rv_r)}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_r}{\partial \theta^2} - \frac{2}{r^2} \frac{\partial v_\theta}{\partial \theta} + \frac{\partial^2 v_r}{\partial z^2} \right] + \rho g_r \quad (3.3.27a)$$

θ direction

$$\rho \left(\frac{\partial v_\theta}{\partial t} + v_r \frac{\partial v_\theta}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_\theta}{\partial \theta} + \frac{v_r v_\theta}{r} + v_z \frac{\partial v_\theta}{\partial z} \right) = -\frac{1}{r} \frac{\partial p}{\partial \theta} + \mu \left[\frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial (rv_\theta)}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_\theta}{\partial \theta^2} + \frac{2}{r^2} \frac{\partial v_r}{\partial \theta} + \frac{\partial^2 v_\theta}{\partial z^2} \right] + \rho g_\theta \quad (3.3.27b)$$

z direction

$$\rho \left(\frac{\partial v_z}{\partial t} + v_r \frac{\partial v_z}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_z}{\partial \theta} + v_z \frac{\partial v_z}{\partial z} \right) = -\frac{\partial p}{\partial z} + \mu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_z}{\partial \theta^2} + \frac{\partial^2 v_z}{\partial z^2} \right] + \rho g_z \quad (3.3.27c)$$

$$0 = \mu \frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) + \rho g_z$$

FOR LIQUID FILM

$$r = R \quad v_z(R) = 0$$

$$r = R - h \quad \left. \frac{\partial v_z}{\partial r} \right|_{r=R-h} = 0$$

ASIDE: IF $\frac{h}{R} \ll 1$ SHOULD APPROX

'FLAT PLATE' SOLUTION FROM ABOVE.

$$0 = \mu \frac{1}{r} \frac{d}{dr} \left(r \frac{dv_z}{dr} \right) + \rho g_z$$

$$\frac{1}{r} \frac{d}{dr} \left(r \frac{dv_z}{dr} \right) = - \frac{\rho g_z}{\mu}$$

$$\frac{d}{dr} \left(r \frac{dv_z}{dr} \right) = - \frac{\rho g_z}{\mu} r$$

$$\int d \left(r \frac{dv_z}{dr} \right) = \int - \frac{\rho g_z}{\mu} r dr$$

$$r \frac{dv_z}{dr} = - \frac{\rho g_z}{\mu} \frac{r^2}{2} + C_1$$

$$\frac{dv_z}{dr} = - \frac{\rho g_z}{\mu} \frac{r}{2} + \frac{C_1}{r}$$

$$\frac{dV_z}{dr} = -\frac{\rho g z}{\mu} \frac{r}{2} + \frac{C_1}{r}$$

$$\int dV_z = \int \left(-\frac{\rho g z}{\mu} \frac{r}{2} + \frac{C_1}{r} \right) dr$$

$$V_z = -\frac{\rho g z}{4\mu} r^2 + C_1 \ln r + C_2$$

FIT BC'S

$$V_z(r) = \frac{\rho g z}{4\mu} \left((R^2 - r^2) + 2(h-R)^2 \ln\left(\frac{r}{R}\right) \right)$$

IF FLOW DOMAIN INCLUDES
 $r=0$, THE "ln" TERM
MUST BE EXCLUDED.

USING MATHEMATICA, THE AVERAGE VELOCITY IS EASILY OBTAINED

Find the average velocity

In[9]:= `Integrate[vzee r, {r, R, R - h}, Assumptions -> {h > 0, h < R}] /
Integrate[r, {r, R, R - h}, Assumptions -> {h > 0, h < R}]`

Out[9]=
$$\frac{g z \rho \left(-\frac{1}{4} h (h - 2 R) (3 h^2 - 6 h R + 2 R^2) + (h - R)^4 \text{Log}\left[1 - \frac{h}{R}\right] \right)}{4 \left(-\frac{R^2}{2} + \frac{1}{2} (-h + R)^2 \right) \mu}$$

In[10]:= `FullSimplify[%]`

Out[10]=
$$\frac{g z \rho \left(-3 h^2 + 6 h R - 2 R^2 + \frac{4 (h - R)^4 \text{Log}\left[1 - \frac{h}{R}\right]}{h (h - 2 R)} \right)}{8 \mu}$$

$Q = \int v_z(r, \theta) dA$



$$\int_R^{R-h} v_z(r) r dr$$

$$= \langle v_z \rangle$$

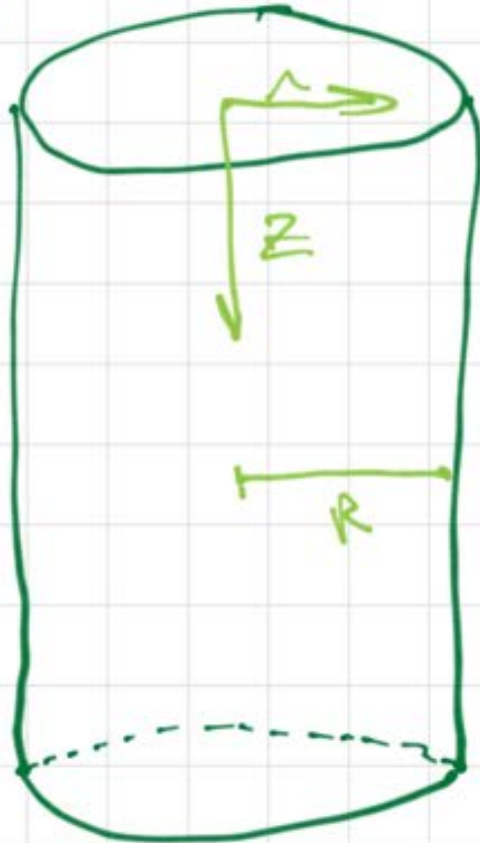
$$\int_R^{R-h} r dr$$

Suppose that we have a circular pipe oriented vertically.

z direction

$$\rho \left(\frac{\partial v_z}{\partial t} + v_r \frac{\partial v_z}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_z}{\partial \theta} + v_z \frac{\partial v_z}{\partial z} \right) = \frac{\partial p}{\partial z} + \mu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_z}{\partial \theta^2} + \frac{\partial^2 v_z}{\partial z^2} \right] + \rho g_z$$

p_1



p_2



$$0 = -\frac{\partial p}{\partial z} + \frac{\mu}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) + \rho g_z$$

BC's

$$v_z(r) = 0$$

$$\left. \frac{\partial v_z}{\partial r} \right|_{r=0} = 0$$

"SYMMETRY"

2 CASES:

1) OPEN AT BOTH ENDS $p_1 = p_2$

2) CLOSE BOTTOM: NO FLOW

LET'S DO CASE #2

$$0 = -\frac{dp}{dz} + \rho g_z$$

$$\frac{dp}{dz} = \rho g_z$$

$$\int dp = \int \rho g_z dz$$

$$p = \rho g_z z + C_0$$

$$p(0) = p_0$$

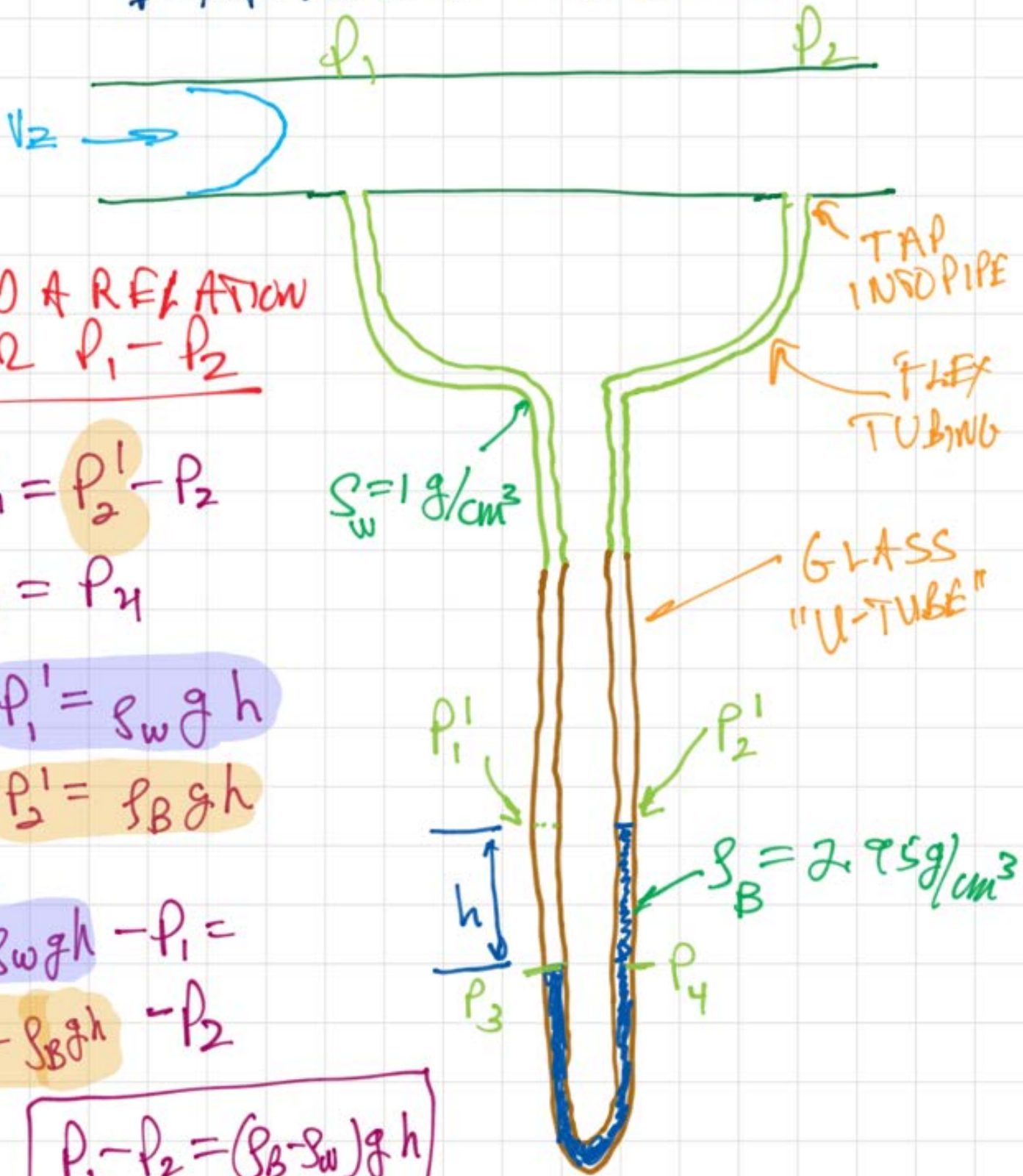
$$\therefore p = p_0 + \rho g_z z$$

p IS LINEAR IN z .

z INCREASES
DOWNWARD

BASIC EQUATION OF
HYDROSTATICS

EXAMPLE: MANOMETER



FIND A RELATION
FOR $p_1 - p_2$

$$p_1' - p_1 = p_2' - p_2$$

$$p_3 = p_4$$

$$p_3 - p_1' = S_w g h$$

$$p_4 - p_2' = S_B g h$$

SUBS:

$$p_3 - S_w g h - p_1 =$$

$$p_4 - S_B g h - p_2$$

\therefore

$$p_1 - p_2 = (S_B - S_w) g h$$

NOW CONSIDER CASE 1:
FLOW BY GRAVITY

$$0 = \frac{\mu}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) + \rho g_z$$

$$\frac{\mu}{r} \frac{d}{dr} \left(r \frac{dv_z}{dr} \right) = -\rho g_z$$

$$\frac{d}{dr} \left(r \frac{dv_z}{dr} \right) = -\frac{\rho}{\mu} g_z r$$

$$\int d \left(r \frac{dv_z}{dr} \right) = -\frac{\rho}{\mu} g_z \int r dr$$

$$r \frac{dv_z}{dr} = -\frac{\rho}{\mu} g_z \frac{r^2}{2} + C_1$$

$$dv_z = \left(-\frac{\rho}{\mu} g_z \frac{r}{2} + \frac{C_1}{r} \right) dr$$

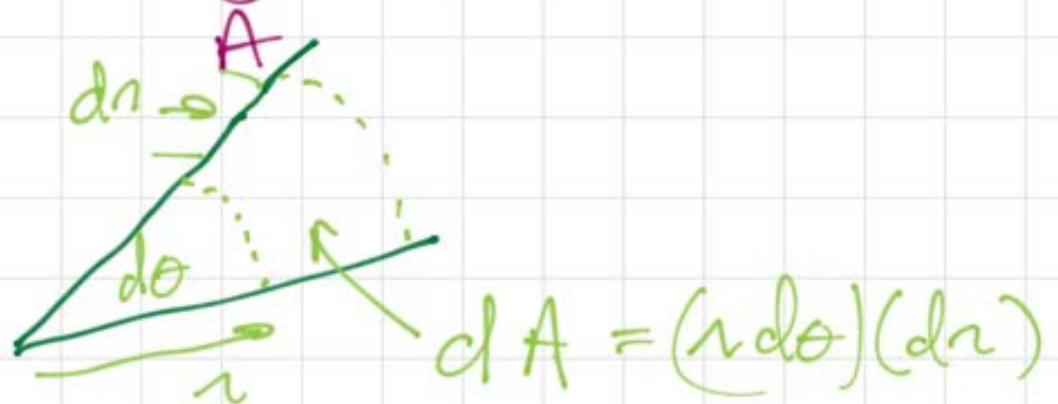
$$v_z = -\frac{\rho}{\mu} g_z \frac{r^2}{4} + C_1 \ln r + C_2$$

BUT $\zeta = 0$
FIT $v_z(r) = 0$

$$v_z(r) = \frac{\rho g z}{4\mu} (R^2 - r^2) \text{ PARABOLIC}$$

WE WOULD LIKE AVERAGE $\langle v_z \rangle$
NEED VOLUMETRIC FLOW

$$Q = \int v_z(r, \theta) dA$$



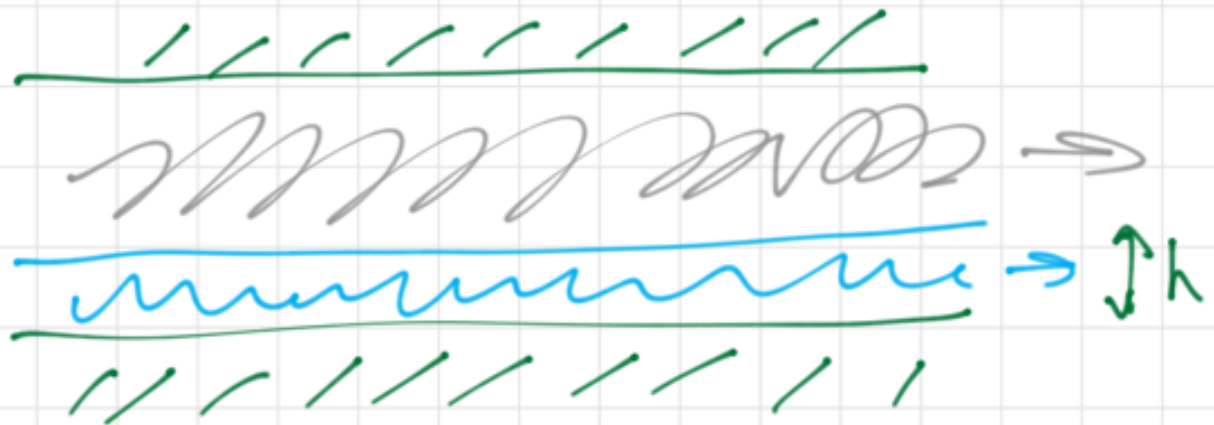
$$\langle v_z \rangle = \frac{\int_0^{2\pi} \int_0^R v_z(r, \theta) r dr d\theta}{\int_0^{2\pi} \int_0^R r dr d\theta}$$
$$\langle v_z \rangle = \frac{\rho g z R^2}{8\mu} = \frac{1}{2} v_{MAX}$$

$$Q = \langle V_z \rangle \pi R^2$$

$$= \frac{2\pi S g_e R^4}{8\mu}$$

VERY SENSITIVE TO THIS VALUE

SUPPOSE 2 IMMISCIBLE OR SEPARATE LAYERS ARE FLOWING TOGETHER



2 SEPARATE EQUATIONS

$$0 = -\frac{\partial p}{\partial x} + \mu^I \frac{\partial^2 V_x^I}{\partial y^2} \quad V_x^I(h) = V_x^II(h)$$

$$0 = -\frac{\partial p}{\partial x} + \mu^{II} \frac{\partial^2 V_x^{II}}{\partial y^2} \quad \mu^I \frac{\partial V_x^I}{\partial y} \Big|_h = \mu^{II} \frac{\partial V_x^{II}}{\partial y}$$

Boundary conditions:

1. Fluid sticks to solid surfaces

$$v_x(0) = 0$$

2. Fluid sticks to another immiscible fluid

$$v_x^I(h) = v_x^{II}(h)$$

3. The shear stress is continuous across the fluid-fluid interface

$$\tau_{yx}^I(h) = \tau_{yx}^{II}(h)$$
$$\mu^I \frac{\partial v_x^I}{\partial y} \Big|_h = \mu^{II} \frac{\partial v_x^{II}}{\partial y}$$

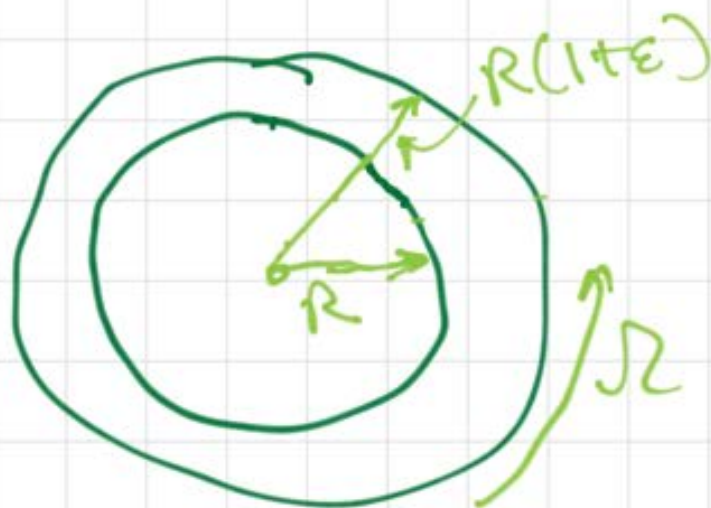
4. For liquid flowing next to quiescent air, the stress can be considered approximately 0.

$$\tau_{yx}(h) \approx 0$$

$$\frac{\partial v_x}{\partial y}(h) \approx 0$$

"FALLING"
FILM

ROTATING FLOW



Cylindrical coordinates

r direction

$$\rho \left(\frac{\partial v_r}{\partial t} + v_r \frac{\partial v_r}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_r}{\partial \theta} - \frac{v_\theta^2}{r} + v_z \frac{\partial v_r}{\partial z} \right) = -\frac{\partial p}{\partial r} + \mu \left[\frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial (rv_r)}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_r}{\partial \theta^2} - \frac{2}{r^2} \frac{\partial v_\theta}{\partial \theta} + \frac{\partial^2 v_r}{\partial z^2} \right] + \rho g_r \quad (3.3.27a)$$

θ direction

$$\rho \left(\frac{\partial v_\theta}{\partial t} + v_r \frac{\partial v_\theta}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_\theta}{\partial \theta} + \frac{v_r v_\theta}{r} + v_z \frac{\partial v_\theta}{\partial z} \right) = -\frac{1}{r} \frac{\partial p}{\partial \theta} + \mu \left[\frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial (rv_\theta)}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_\theta}{\partial \theta^2} + \frac{2}{r^2} \frac{\partial v_r}{\partial \theta} + \frac{\partial^2 v_\theta}{\partial z^2} \right] + \rho g_\theta \quad (3.3.27b)$$

z direction

$$\rho \left(\frac{\partial v_z}{\partial t} + v_r \frac{\partial v_z}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_z}{\partial \theta} + v_z \frac{\partial v_z}{\partial z} \right) = -\frac{\partial p}{\partial z} + \mu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_z}{\partial \theta^2} + \frac{\partial^2 v_z}{\partial z^2} \right] + \rho g_z \quad (3.3.27c)$$

$$0 = \mu \frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial (rv_\theta)}{\partial r} \right)$$

$$v_\theta(R) = 0$$

$$v_\theta(R(1+\epsilon)) = R(1+\epsilon)\Omega$$

SOLVE

$$\frac{1}{r} \frac{d}{dr} (rv_\theta) = C_1$$

$$\frac{d}{dr} (rv_\theta) = \frac{C_1}{r}$$

$$v_\theta = \frac{C_1}{2} + \frac{C_2}{r}$$

NO $r=0$
SO
BOTH
TERMS
ARE OK.

FIT B.C.'S

$$0 = \frac{C_1 R}{2} + \frac{C_2}{R}$$

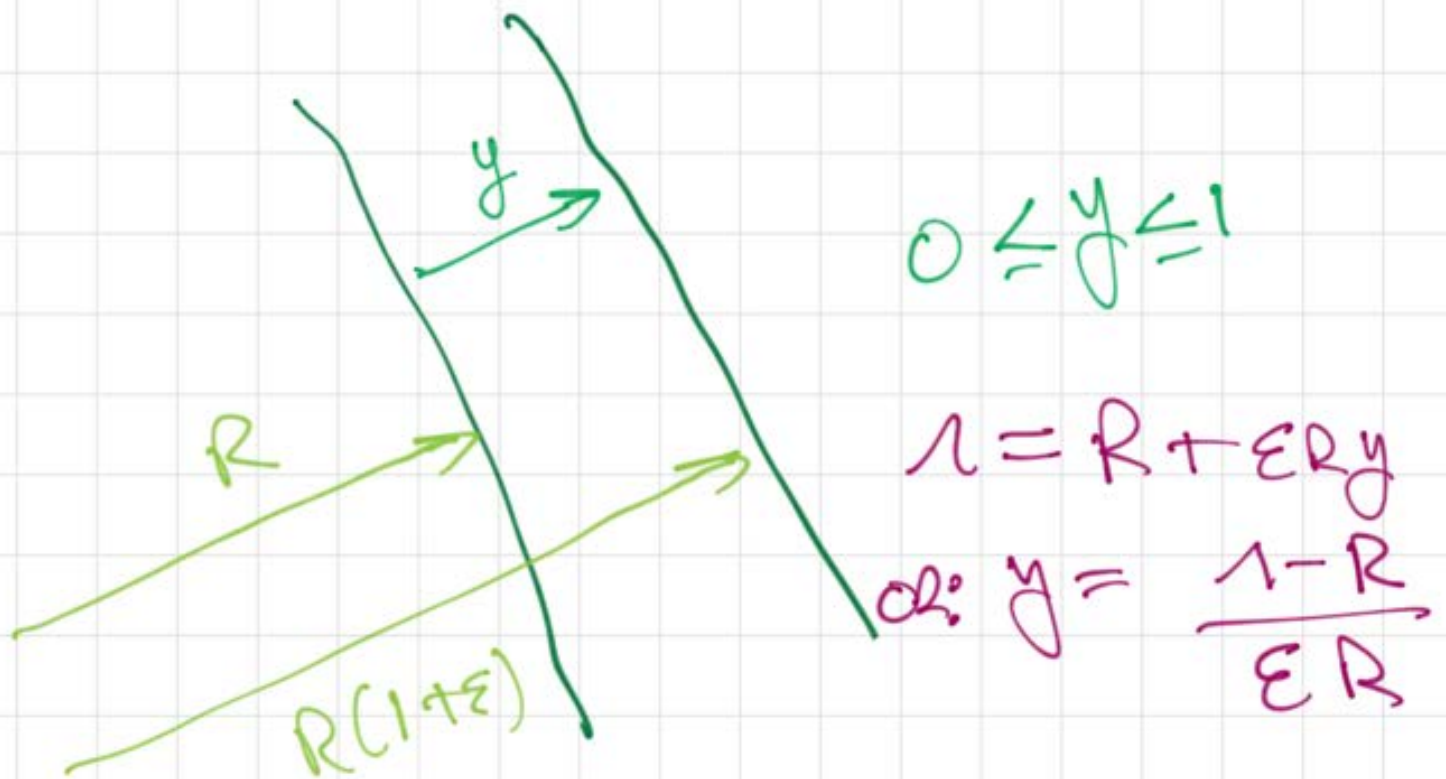
$$C_2 = -\frac{C_1 R^2}{2}$$

$$R(1+\epsilon)\Omega = C_1 \frac{R(1+\epsilon)}{2} - \frac{C_1 R^2}{2R(1+\epsilon)}$$

$$1 = \frac{C_1}{2\Omega} \left(1 - \frac{1}{(1+\epsilon)^2} \right)$$

$$C_1 = \frac{2\Omega}{\left(1 - \frac{1}{(1+\epsilon)^2} \right)}$$

$$V_{\theta}(\lambda) = \frac{\lambda}{\left(1 - \frac{1}{(1+\epsilon)^2}\right)} \left(\lambda - \frac{R^2}{\lambda}\right)$$



SERIES EXPANSION IN ϵ

$$V_{\theta}(y) = R \lambda y + \frac{\epsilon}{2} \left(\frac{3Ry\lambda^2}{2} - \frac{1}{2} R y^3 \lambda \right)$$

AS $\epsilon \rightarrow 0$ LOOKS LIKE
PARALLEL PLATE

USING WOLFRAM ALPHA TO GET SERIES EXPANSION

7:37 AM

81%

WolframAlpha

substitute $r=R+R*\epsilon*y$ into $(r-R)*(r+R)*(1+\epsilon)^2*\omega/r/\epsilon/(2+\epsilon)$

Input interpretation

$$(r - R)(r + R)(1 + \epsilon)^2 \times \frac{r}{2 + \epsilon} \text{ where } r = R + R y \epsilon$$

Result

$$\frac{R y \omega (\epsilon + 1)^2 (R y \epsilon + 2 R)}{(\epsilon + 2) (R y \epsilon + R)}$$

Step-by-step solution

☰

WolframAlpha

📄

taylor series $R*y*\omega*(\epsilon+1)^2*(R*y*\epsilon+2*R)/(\epsilon+2)/(R*y*\epsilon+R)$ in ϵ

Input interpretation

series	$R y \omega (\epsilon + 1)^2 \times \frac{R y \epsilon + 2 R}{R y \epsilon + R}$	point	$\epsilon = 0$
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Series expansion at $\epsilon=0$

$$R y \omega - \frac{1}{2} \epsilon (R (y - 3) y \omega) + \frac{1}{4} R y (2 y^2 - 3 y + 1) \omega \epsilon^2 - \frac{1}{8} \epsilon^3 (R y (4 y^3 - 6 y^2 + y + 1) \omega) + \frac{1}{16} R y (8 y^4 - 12 y^3 + 2 y^2 + y + 1) \omega \epsilon^4 - \frac{1}{32} \epsilon^5 (R y (16 y^5 - 24 y^4 + 4 y^3 + 2 y^2 + y + 1) \omega) + O(\epsilon^6)$$

(Taylor series)

$$\lim_{\epsilon \rightarrow 0} V = y(R\omega) \quad 0 \leq y \leq 1$$

CATHETER FLOW

8:00 AM

76%

WolframAlpha

$\text{dsolve}(\mu/r*(v'(r)+r*v''(r))-G=0, v(R)=0, v(\lambda R)=0)$

Input

$\left\{ \frac{\mu}{r} (v'(r) + r v''(r)) - G = 0, v(R) = 0, v(\lambda R) = 0 \right\}$

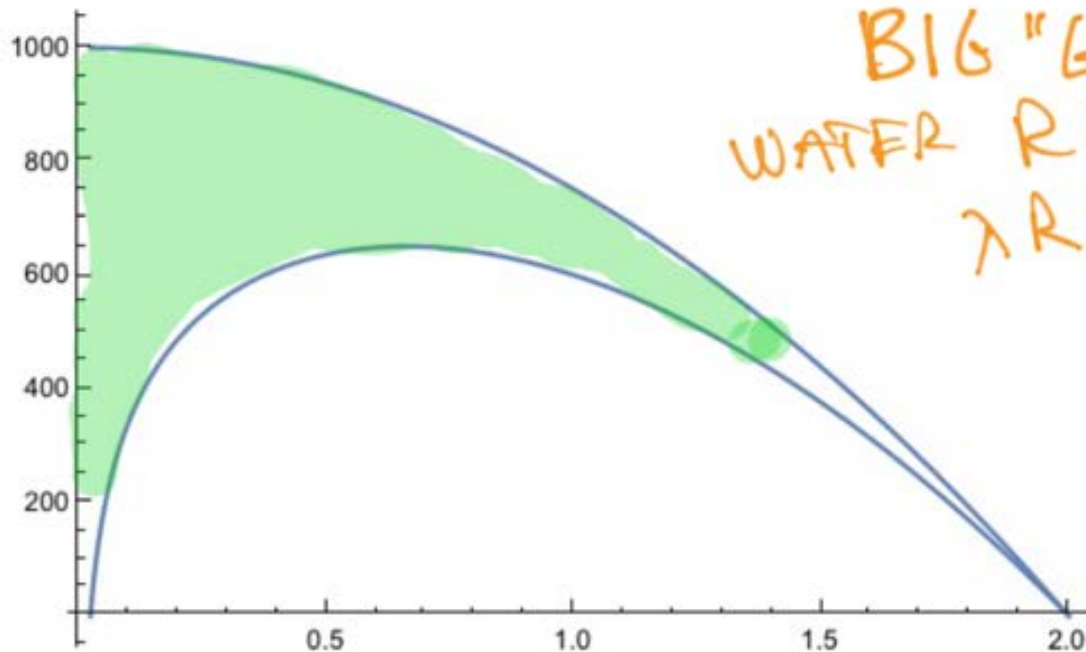
Differential equation solution

$$v(r) = \frac{G (\log(R) (r^2 - \lambda^2 R^2) + (R^2 - r^2) \log(\lambda R) + (\lambda^2 - 1) R^2 \log(r))}{4 \mu (\log(R) - \log(\lambda R))}$$

Step-by-step solution

Related Wolfram|Alpha queries

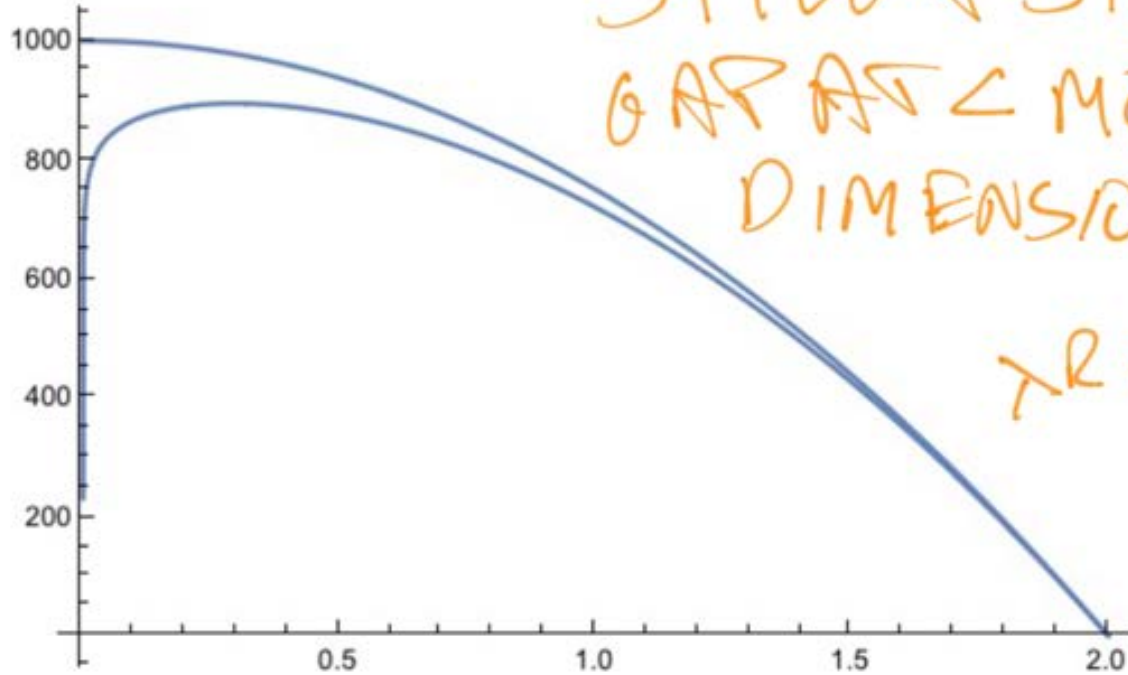
$\text{Plot}\{v_{eez}, v_{zee}\} /. \{d_{pdz} \rightarrow -10, \mu \rightarrow .01, R \rightarrow 2, \lambda \rightarrow .01\}, \{r, .02, 2\}$



Plot[{veez, vezee} /. {dpdz → -10, μ → .01, R → 2, λ → .00000000001},
 {r, .00000000002, 2}]

STILL A SIGNIFICANT
 GAP AS $\lambda <$ MOLECULAR
 DIMENSION !!

$$\lambda R = 2 \times 10^{-10} \text{ cm}$$



LIMIT OF OPEN CHANNEL
 EXISTS, BUT NOT RELEVANT
 PHYSICALLY

In[36]:= Series[veez, {λ, 0, 2}]

$$\text{Out[36]} = \frac{1}{4 \mu \text{Log}[\lambda]} (\text{dpdz } R^2 \text{Log}[r] - \text{dpdz } R^2 \text{Log}[R] + \text{dpdz } r^2 \text{Log}[\lambda] - \text{dpdz } R^2 \text{Log}[\lambda]) +$$

$$\frac{(-\text{dpdz } R^2 \text{Log}[r] + \text{dpdz } R^2 \text{Log}[R]) \lambda^2}{4 \mu \text{Log}[\lambda]} + O[\lambda]^3$$

In[37]:= Apart[Normal[%]]

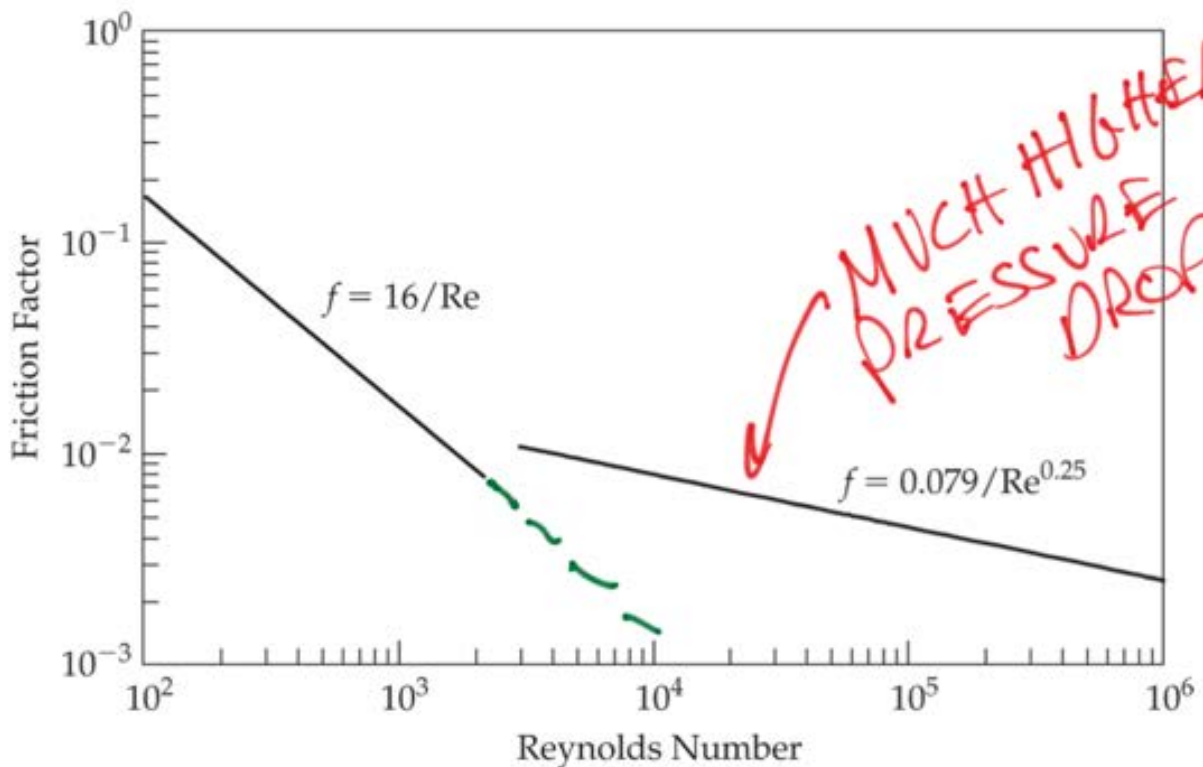
$$\text{Out[37]} = -\frac{\text{dpdz } (-r^2 + R^2)}{4 \mu} \frac{1}{4 \mu \text{Log}[\lambda]}$$

$$(\text{dpdz } R^2 \text{Log}[r] - \text{dpdz } R^2 \lambda^2 \text{Log}[r] - \text{dpdz } R^2 \text{Log}[R] + \text{dpdz } R^2 \lambda^2 \text{Log}[R])$$

TRANSITION TO TURBULENT FLOW.

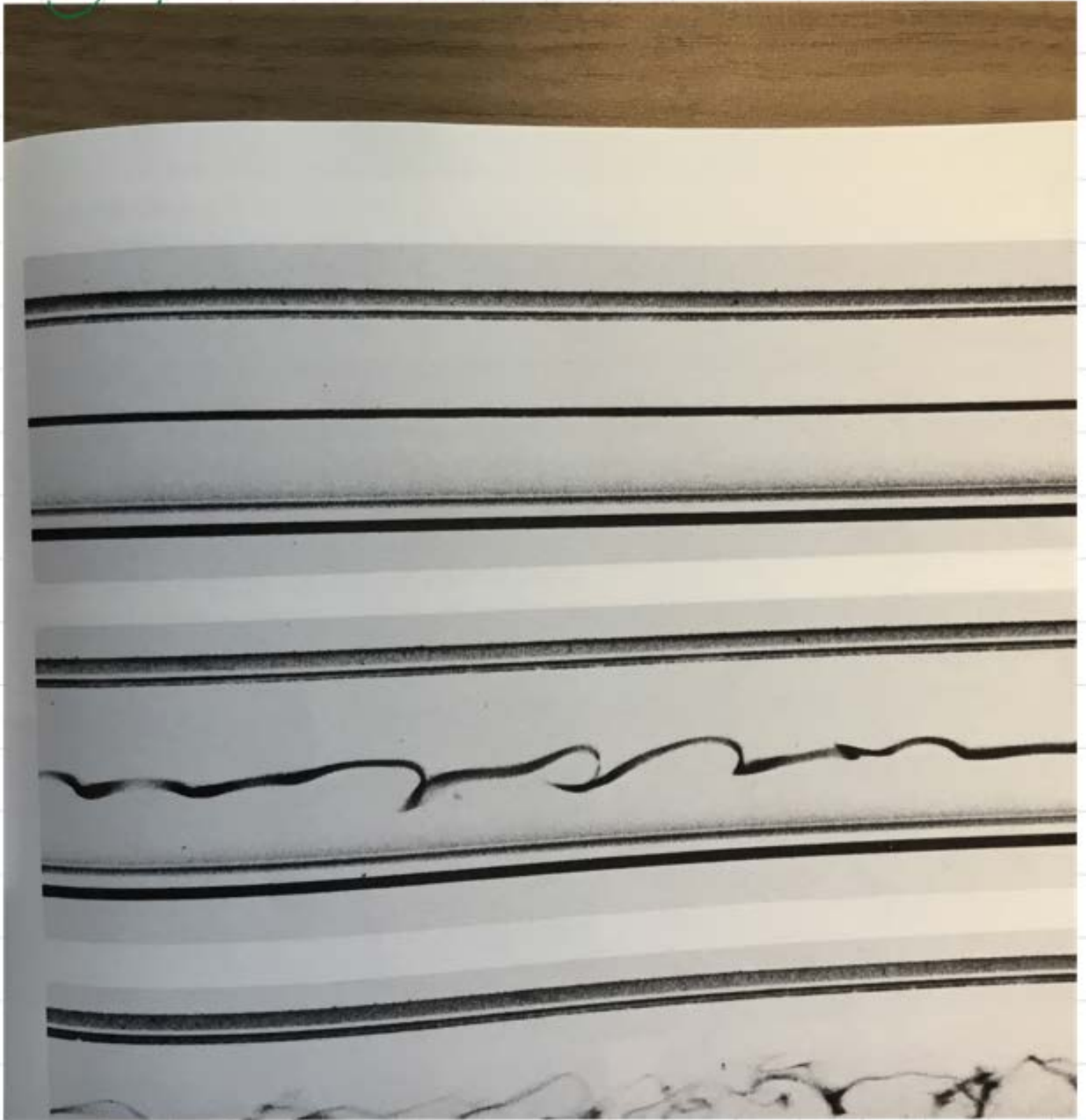
$$Re \equiv \frac{DVS}{\mu}$$

Figure 3.11 The Fanning friction factor versus the Reynolds number.



$$f \equiv \frac{\Delta P D}{2 L \rho V^2}$$

DYE STREAM IN PIPE FLOW





103. Repetition of Reynolds' dye experiment. Osborne Reynolds' celebrated 1883 investigation of stability of flow in a tube was documented by sketches rather than photography. However the original apparatus has survived at the University of Manchester. Using it a century later, N. H. Johannesen and C. Lowe have taken this sequence of photographs. In laminar flow a filament of colored water

introduced at a bell-shaped entry extends undisturbed the whole length of the glass tube. Transition is seen in the second of the photographs as the speed is increased; and the last two photographs show fully turbulent flow. Modern traffic in the streets of Manchester made the critical Reynolds number lower than the value 13,000 found by Reynolds.

61

AS $Re \uparrow$ MORE "MIXING"
STRONGER DISTURBANCES