

CBE 30357

9/12/17

TOPICS FOR TODAY

1) REVIEW OF SLIDING
SURFACE FLOW

2) FLOWS CAUSED BY A
PRESSURE GRADIENT

3) FLOWS CAUSED BY
GRAVITY

* ALONG WITH "WIND", THE PRIMARY
ACTIONS OR FORCES THAT CAUSE
FLUID FLOW.

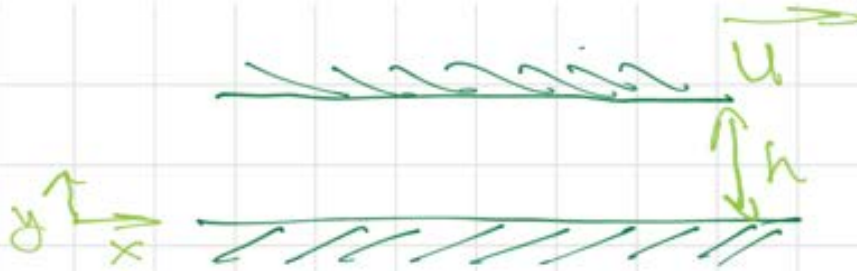


TABLE 3.4

Navier-Stokes Equation for an Incompressible Fluid

Rectangular coordinates

x direction

$$\rho \left(\frac{\partial v_x}{\partial t} + v_x \frac{\partial v_x}{\partial x} + v_y \frac{\partial v_x}{\partial y} + v_z \frac{\partial v_x}{\partial z} \right) = -\frac{\partial p}{\partial x} + \mu \left[\frac{\partial^2 v_x}{\partial x^2} + \frac{\partial^2 v_x}{\partial y^2} + \frac{\partial^2 v_x}{\partial z^2} \right] + \rho g_x$$

y direction

$$\rho \left(\frac{\partial v_y}{\partial t} + v_x \frac{\partial v_y}{\partial x} + v_y \frac{\partial v_y}{\partial y} + v_z \frac{\partial v_y}{\partial z} \right) = -\frac{\partial p}{\partial y} + \mu \left[\frac{\partial^2 v_y}{\partial x^2} + \frac{\partial^2 v_y}{\partial y^2} + \frac{\partial^2 v_y}{\partial z^2} \right] + \rho g_y$$

z direction

$$\rho \left(\frac{\partial v_z}{\partial t} + v_x \frac{\partial v_z}{\partial x} + v_y \frac{\partial v_z}{\partial y} + v_z \frac{\partial v_z}{\partial z} \right) = -\frac{\partial p}{\partial z} + \mu \left[\frac{\partial^2 v_z}{\partial x^2} + \frac{\partial^2 v_z}{\partial y^2} + \frac{\partial^2 v_z}{\partial z^2} \right] + \rho g_z$$

AFTER SOME WORK

$$0 = \mu \frac{\partial^2 v_x}{\partial y^2}$$

BOUNDARY CONDITIONS..

$$v_x(0) = 0, \quad v_x(h) = u$$

FLUID STICKS TO SOLID SURFACES!

$$v_x = C_1 y + C_2$$

$$v_x(y) = \frac{y}{h} u$$

WHICH STRESS COMPONENTS ARE IMPORTANT??

$$v_x(y) = \frac{y}{h} U \quad \leftarrow \text{NO VISCOSITY}$$

$$\tau_{yx} = \mu \frac{\partial v_x}{\partial y} = \frac{\mu U}{h} \quad \leftarrow \text{VISCOSITY IN STRESS}$$

CONSTANT, NOT A FUNCTION OF y

$$\text{FORCE } F_x = \tau_{yx} \times W \times L$$

(FORCE/AREA)

WIDTH IN z LENGTH IN x

$$= \mu \frac{U}{h} W L$$

MORE VISCIOUS $\uparrow \mu$ FASTER $\uparrow U$ CLOSER SPACING $\uparrow F$

AVERAGE VELOCITY = $\frac{\text{TOTAL VOLUMETRIC FLOW}}{\text{AREA OF FLOW}}$

$$\langle v_x \rangle = \frac{\int_0^w \int_0^h v_x(y) dy dz}{\int_0^w \int_0^h dy dz}$$

$$\langle v_x \rangle = \frac{W \frac{U}{h} \frac{y^2}{2} \Big|_0^h}{Wh} = \frac{U}{2}$$

SUPPOSE WE KEEP THE
FLOW GEOMETRY THE
SAME, BUT CAUSE
THE FLOW BY A
PRESSURE GRADIENT

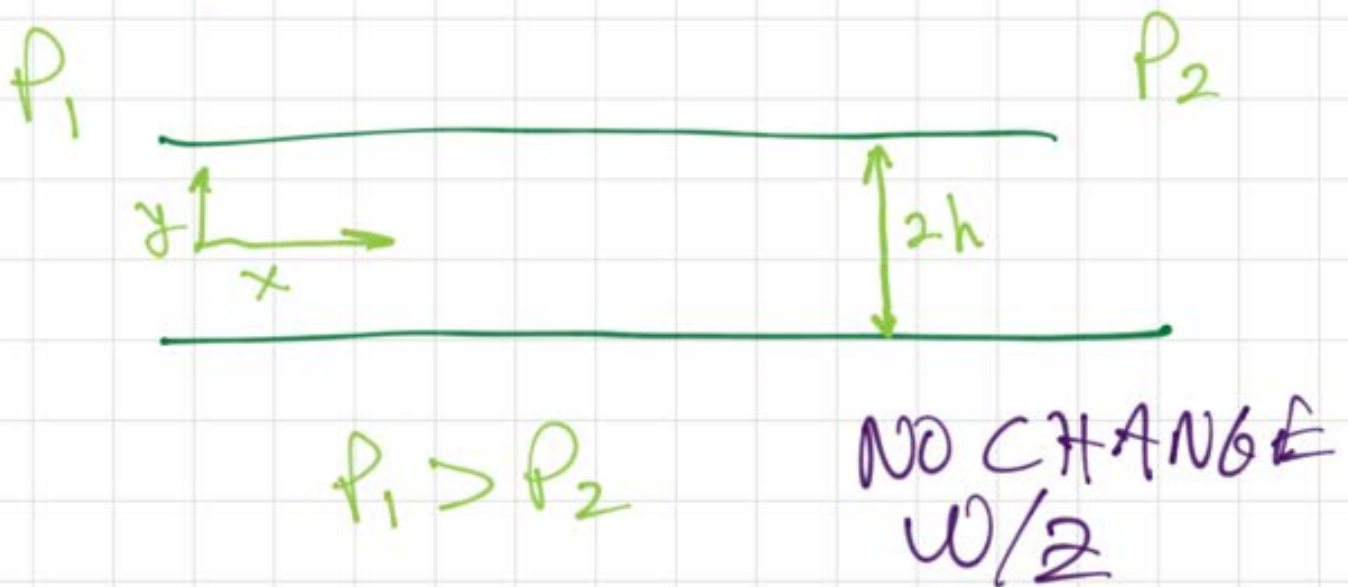


TABLE 3.4

Navier-Stokes Equation for an Incompressible Fluid

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y direction

$$\rho \left(\frac{\partial v_y}{\partial t} + v_x \frac{\partial v_y}{\partial x} + v_y \frac{\partial v_y}{\partial y} + v_z \frac{\partial v_y}{\partial z} \right) = -\frac{\partial p}{\partial y} + \mu \left[\frac{\partial^2 v_y}{\partial x^2} + \frac{\partial^2 v_y}{\partial y^2} + \frac{\partial^2 v_y}{\partial z^2} \right] + \rho g_y$$

z direction

$$\rho \left(\frac{\partial v_z}{\partial t} + v_x \frac{\partial v_z}{\partial x} + v_y \frac{\partial v_z}{\partial y} + v_z \frac{\partial v_z}{\partial z} \right) = -\frac{\partial p}{\partial z} + \mu \left[\frac{\partial^2 v_z}{\partial x^2} + \frac{\partial^2 v_z}{\partial y^2} + \frac{\partial^2 v_z}{\partial z^2} \right] + \rho g_z$$

$$0 = -\frac{\partial p}{\partial x} + \mu \frac{\partial^2 v_x}{\partial y^2}$$

BOUNDARY CONDITIONS:

$$v_x(h) = 0$$

$$v_x(-h) = 0$$

[NO
SLIP]

WE SOLVE:

$$\frac{d^2 v_x}{dy^2} = \frac{1}{\mu} \frac{dp}{dx}$$

A subtle and perhaps confusing argument will now be given! 🤔

We conclude that v_x does not change in x .

We conclude that $\frac{\partial p}{\partial x}$ does not change with y

Hence, the only way for these terms to be always equal is if they are equal to a constant.

$$\frac{d^2 v_x}{dy^2} = \frac{1}{\mu} \frac{dp}{dx} = \text{CONSTANT}$$

NOW WE PROCEED...

$$\frac{d}{dy} \left(\frac{dv_x}{dy} \right) = \frac{1}{\mu} \frac{dp}{dx}$$

$$d \left(\frac{dv_x}{dy} \right) = \frac{1}{\mu} \frac{dp}{dx} dy$$

$$\int d \left(\frac{dv_x}{dy} \right) = \int \frac{1}{\mu} \frac{dp}{dx} dy$$

$$\frac{dV_x}{dy} = \frac{1}{\mu} \frac{dP}{dx} y + C_1$$

$$dV_x = \left(\frac{1}{\mu} \frac{dP}{dx} y + C_1 \right) dy$$

$$\int dV_x = \int \left(\frac{1}{\mu} \frac{dP}{dx} y + C_1 \right) dy$$

$$V_x = \frac{1}{\mu} \frac{dP}{dx} \frac{y^2}{2} + C_1 y + C_2$$

NOW WE NEED TO APPLY
BOUNDARY CONDITIONS

$$V_x(h) = 0 = \frac{1}{\mu} \frac{dP}{dx} \frac{h^2}{2} + C_1 h + C_2$$

$$V_x(-h) = 0 = \frac{1}{\mu} \frac{dP}{dx} \frac{h^2}{2} - C_1 h + C_2$$

$$\therefore C_1 = 0$$

$$C_2 = -\frac{1}{\mu} \frac{dP}{dx} \frac{h^2}{2}$$

WHICH GIVES

$$v_x(y) = -\frac{1}{\mu} \frac{dP}{dx} \frac{h^2}{2} \left(1 - \frac{y^2}{h^2} \right)$$

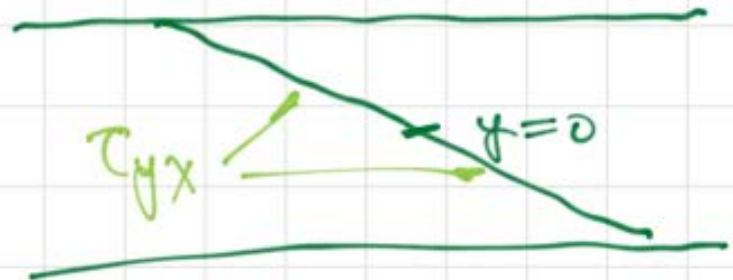


SHEAR STRESS

$$\tau_{yx} = \mu \frac{\partial v_x}{\partial y}$$

$$= \frac{dP}{dx} y$$

(-)



STRESS IS MAXIMUM AT WALLS, 0 IN MIDDLE

WHAT IS AVERAGE VELOCITY?

$$\langle v_x \rangle = \frac{\int_0^w \int_{-h}^h v_x(y) dy dz}{\int_0^w \int_{-h}^h dy dz}$$

$$v_x(y) = -\frac{1}{\mu} \frac{dp}{dx} \frac{h^2}{2} \left(1 - \frac{y^2}{h^2} \right)$$

$$\langle v_x \rangle = \frac{-\frac{1}{\mu} \frac{dp}{dx} \frac{h^2}{2} w}{2wh} \int_{-h}^h \left(1 - \frac{y^2}{h^2} \right) dy$$

$$= \frac{-\frac{1}{\mu} \frac{dp}{dx} \frac{h^2 w}{2}}{2wh} \left(y - \frac{y^3}{3h^2} \right) \Big|_{-h}^h$$

$$= \frac{-1}{4\mu} \frac{dp}{dx} h \left(\left(h - \frac{1}{3}h \right) - \left(-h + \frac{1}{3}h \right) \right)$$

$$\langle v_x \rangle = \frac{1}{3\mu} \left(-\frac{dp}{dx} \right) h^2$$

PRESSURE "DROPS" LINEARLY WITH DISTANCE, HENCE

$$\frac{dp}{dx} < 0$$

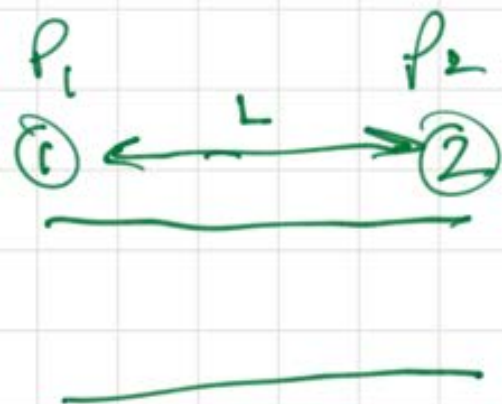
$$\frac{d^2 v_x}{dy^2} = \frac{1}{\mu} \frac{dp}{dx} = \text{CONSTANT}$$

$$\frac{dp}{dx} = C_1$$

$$dp = C_1 dx$$

$$\int dp = \int C_1 dx$$

$$p = C_1 x + C_2$$



$$\textcircled{1} x=0, P=P_1$$

$$\therefore P = C_1 x + P_1$$

$$\text{IF } P = P_2 \text{ @ } x = L$$

$$P_2 = C_1 L + P_1$$

$$C_1 = \frac{P_2 - P_1}{L}$$

OR

$$P = \left[\frac{P_2 - P_1}{L} \right] x + P_1$$
$$= \frac{dP}{dx} L \textcircled{1}$$

PRESSURE DECREASES
LINEARLY WITH DISTANCE

How does gravity cause fluid flow?



X-DIRECTION NAVIER-STOKES EQ.

$$0 = \mu \frac{d^2 v_x}{dy^2} + \rho g \cos \theta$$

BOUNDARY CONDITIONS

$$v_x(0) = 0$$

AT $y = h$ WE HAVE TO FIND
SOMETHING ELSE.....

$$\tau_{yx}(y=h) = 0 \quad \text{NO STRESS} \quad \text{📞}$$

$$0 = \mu \frac{d^2 v_x}{dy^2} + \rho g \cos \theta$$

$$\frac{d^2 v_x}{dy^2} = -\frac{\rho g}{\mu} \cos \theta$$

$$\frac{d}{dy} \left(\frac{dv_x}{dy} \right) = -\frac{\rho g}{\mu} \cos \theta$$

$$\int d \left(\frac{dv_x}{dy} \right) = -\frac{\rho g}{\mu} \cos \theta \int dy$$

$$\frac{dv_x}{dy} = -\frac{\rho g}{\mu} \cos \theta y + C_1$$

$$\int dv_x = \int \left(-\frac{\rho g}{\mu} \cos \theta y + C_1 \right) dy$$

$$v_x = -\frac{\rho g}{\mu} \cos \theta \frac{y^2}{2} + C_1 y + C_2$$

$$\textcircled{1} \quad y=0 \quad v_x(0)=0$$

$$\therefore C_2 = 0$$

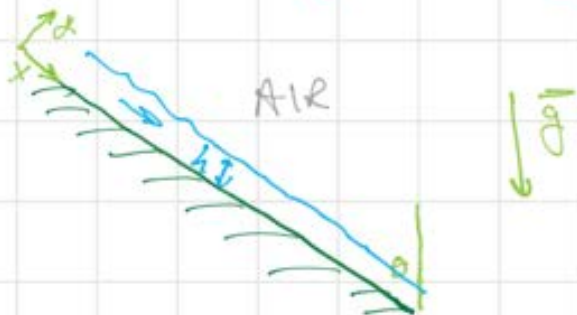
$$\textcircled{2} \quad y=h \quad \frac{dv_x}{dy} = 0$$

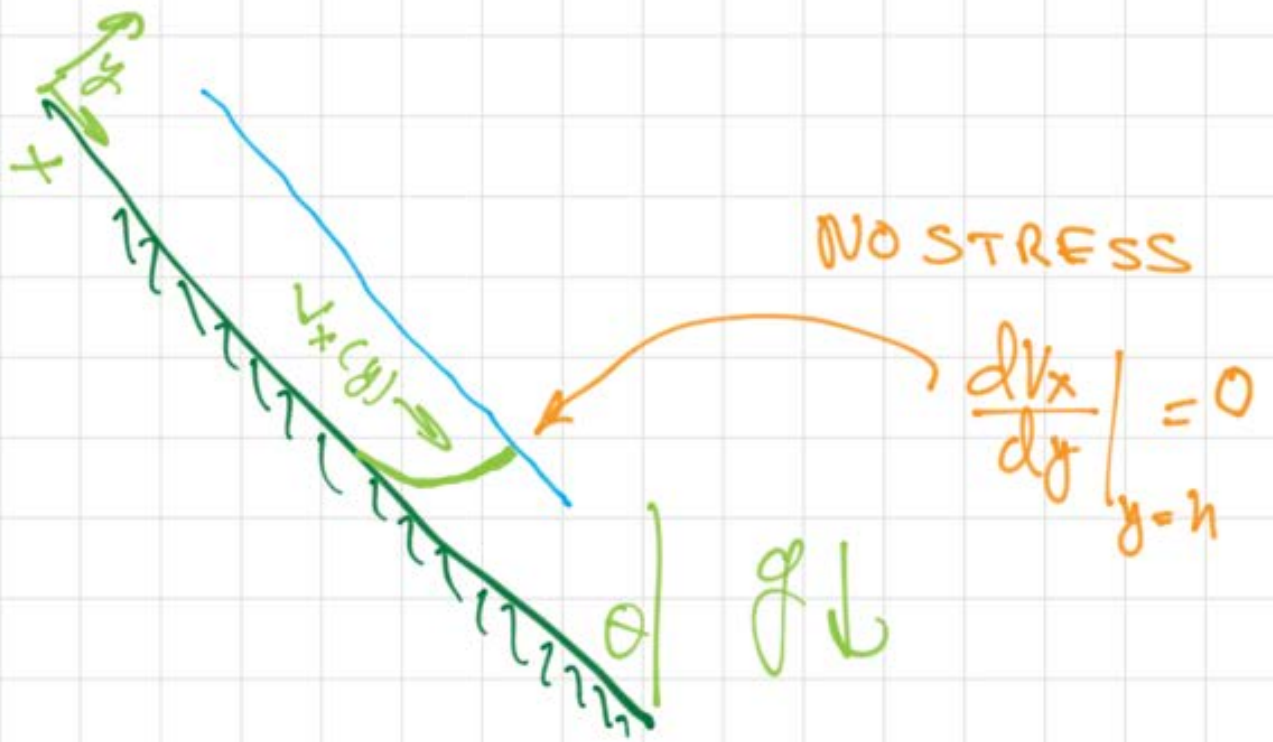
$$\frac{dv_x}{dy} = -\frac{\rho g}{\mu} \cos \theta y + C_1$$

$$\therefore C_1 = \frac{\rho g}{\mu} \cos \theta h$$

THUS THE PROFILE IS

$$v_x(y) = \frac{\rho g}{\mu} \cos \theta \left(\frac{y^2}{2} - yh \right)$$





The mechanism of exercise-induced asthma is . . .

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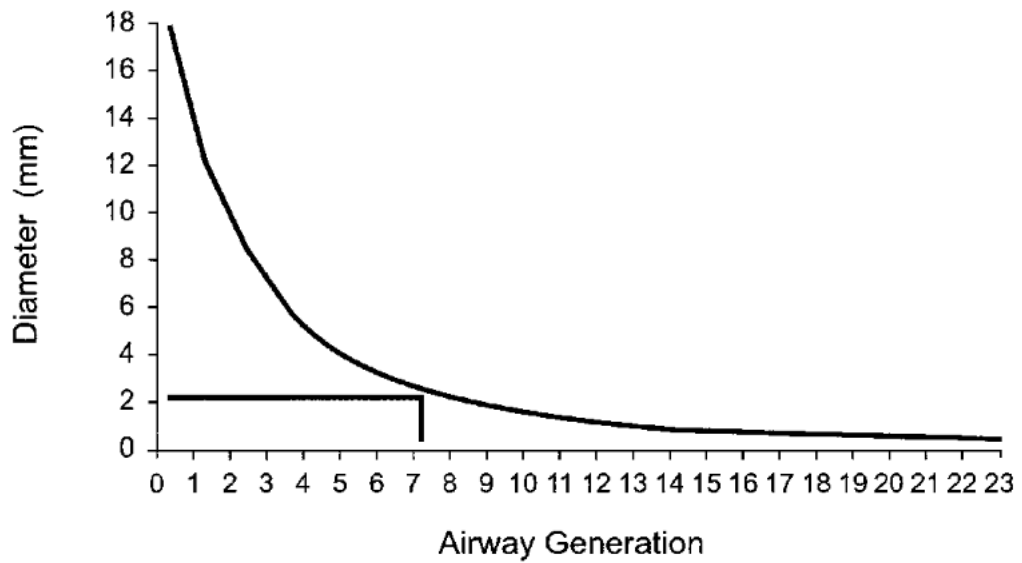


Figure 1. Airway diameters (mm) drawn against airway generations. (Adapted with permission from Ref. 6.)

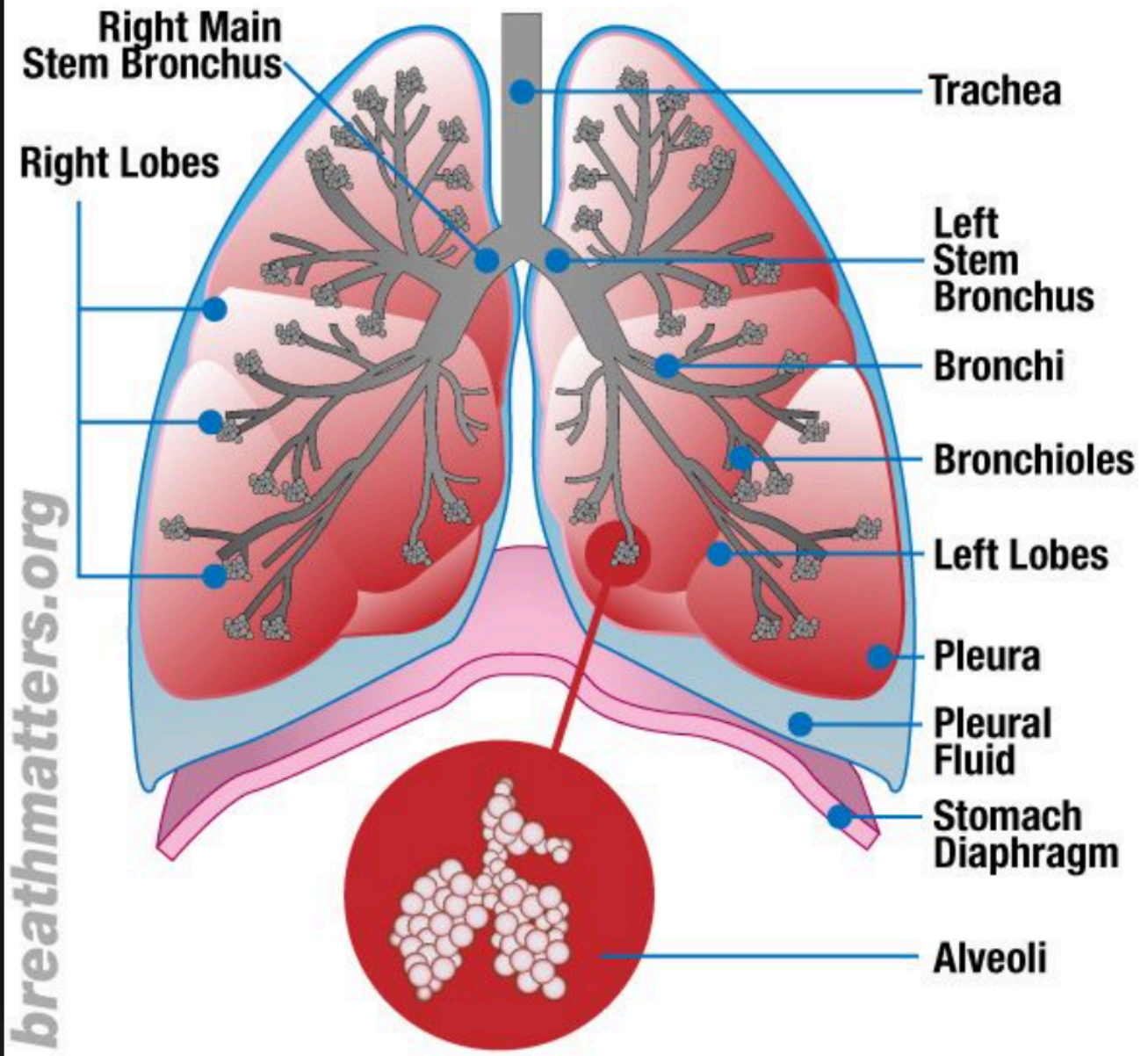
MECHANISMS OF REMODELLING

perhaps best illustrated by irritant-induced asthma. Certain characteristic changes, such as growth of ASM, seem likely to contribute to airways hyperresponsiveness (AHR), but the consequences of other changes, such as subepithelial fibrosis, are not so intuitively obvious. Even the link between growth of ASM and AHR is deduced largely on the basis of modelling studies. One of the major challenges facing the field is to devise strategies that link structure and function directly. For



Airway smooth
muscle (remodeling)

LUNGS



breathmatters.org

Suppose we want to consider the flow in a trachea.

The geometry is a cylinder.

We need the Navier-Stokes equations in cylindrical coordinates.

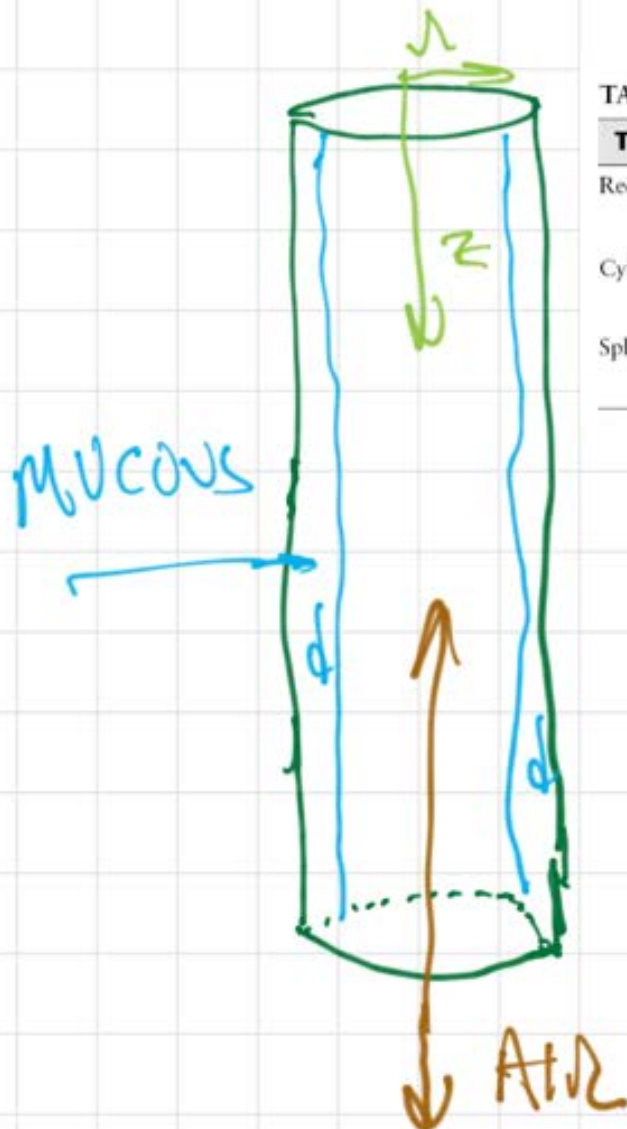


TABLE 3.1

The Conservation of Mass (Continuity Equation)

Rectangular coordinates (x, y, z)	$\frac{\partial \rho}{\partial t} = -\left(\frac{\partial(\rho v_x)}{\partial x} + \frac{\partial(\rho v_y)}{\partial y} + \frac{\partial(\rho v_z)}{\partial z}\right)$
Cylindrical coordinates (r, θ, z)	$\frac{\partial \rho}{\partial t} = -\left(\frac{1}{r} \frac{\partial(\rho r v_r)}{\partial r} + \frac{1}{r} \frac{\partial(\rho v_\theta)}{\partial \theta} + \frac{\partial(\rho v_z)}{\partial z}\right)$
Spherical coordinates (r, θ, ϕ)	$\frac{\partial \rho}{\partial t} = -\left(\frac{1}{r^2} \frac{\partial(\rho r^2 v_r)}{\partial r} + \frac{1}{r \sin \theta} \frac{\partial(\rho v_\theta \sin \theta)}{\partial \theta} + \frac{1}{r \sin \theta} \frac{\partial(\rho v_\phi)}{\partial \phi}\right)$

$$\frac{\partial v_z}{\partial z} = 0$$

$$v_z = v_z(r)$$

ONLY

Cylindrical coordinates

r direction

$$\rho \left(\frac{\partial v_r}{\partial t} + v_r \frac{\partial v_r}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_r}{\partial \theta} - \frac{v_\theta^2}{r} + v_z \frac{\partial v_r}{\partial z} \right) = -\frac{\partial p}{\partial r} + \mu \left[\frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial (r v_r)}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_r}{\partial \theta^2} - \frac{2}{r^2} \frac{\partial v_\theta}{\partial \theta} + \frac{\partial^2 v_r}{\partial z^2} \right] + \rho g_r \quad (3.3.27a)$$

θ direction

$$\rho \left(\frac{\partial v_\theta}{\partial t} + v_r \frac{\partial v_\theta}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_\theta}{\partial \theta} + \frac{v_r v_\theta}{r} + v_z \frac{\partial v_\theta}{\partial z} \right) = -\frac{1}{r} \frac{\partial p}{\partial \theta} + \mu \left[\frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial (r v_\theta)}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_\theta}{\partial \theta^2} + \frac{2}{r^2} \frac{\partial v_r}{\partial \theta} + \frac{\partial^2 v_\theta}{\partial z^2} \right] + \rho g_\theta \quad (3.3.27b)$$

z direction

$$\rho \left(\frac{\partial v_z}{\partial t} + v_r \frac{\partial v_z}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_z}{\partial \theta} + v_z \frac{\partial v_z}{\partial z} \right) = -\frac{\partial p}{\partial z} + \mu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_z}{\partial \theta^2} + \frac{\partial^2 v_z}{\partial z^2} \right] + \rho g_z \quad (3.3.27c)$$

$$0 = \mu \frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) + \rho g_z$$

FOR LIQUID FILM

$$r = R \quad v_z(R) = 0$$

$$r = R - h \quad \left. \frac{\partial v_z}{\partial r} \right|_{r=R-h} = 0$$

SEEMS LIKE ESSENTIALLY
SAME PROBLEM IF $\frac{h}{R} \ll 1$

$$0 = \mu \frac{1}{r} \frac{d}{dr} \left(r \frac{dv_z}{dr} \right) + \rho g_z$$

$$\frac{1}{r} \frac{d}{dr} \left(r \frac{dv_z}{dr} \right) = - \frac{\rho g_z}{\mu}$$

$$\frac{d}{dr} \left(r \frac{dv_z}{dr} \right) = - \frac{\rho g_z}{\mu} r$$

$$\int d \left(r \frac{dv_z}{dr} \right) = \int - \frac{\rho g_z}{\mu} r dr$$

$$r \frac{dv_z}{dr} = - \frac{\rho g_z}{\mu} \frac{r^2}{2} + C_1$$

$$\frac{dv_z}{dr} = - \frac{\rho g_z}{\mu} \frac{r}{2} + \frac{C_1}{r}$$

$$\frac{dV_z}{dr} = -\frac{\rho g z}{\mu} \frac{r}{2} + \frac{C_1}{r}$$

$$\int dV_z = \int \left(-\frac{\rho g z}{\mu} \frac{r}{2} + \frac{C_1}{r} \right) dr$$

$$V_z = -\frac{\rho g z}{4\mu} r^2 + C_1 \ln r + C_2$$

FIT BC'S

$$V_z(r) = \frac{\rho g z}{4\mu} \left((R^2 - r^2) + 2(h-R)^2 \ln\left(\frac{r}{R}\right) \right)$$

IF FLOW DOMAIN INCLUDES
 $r=0$, THE "ln" TERM
 MUST BE EXCLUDED.

USING MATHEMATICA, THE AVERAGE VELOCITY IS EASILY OBTAINED

Find the average velocity

In[9]:= `Integrate[vzee r, {r, R, R - h}, Assumptions -> {h > 0, h < R}] /
Integrate[r, {r, R, R - h}, Assumptions -> {h > 0, h < R}]`

Out[9]=
$$\frac{g z \rho \left(-\frac{1}{4} h (h - 2 R) (3 h^2 - 6 h R + 2 R^2) + (h - R)^4 \operatorname{Log}\left[1 - \frac{h}{R}\right] \right)}{4 \left(-\frac{R^2}{2} + \frac{1}{2} (-h + R)^2 \right) \mu}$$

In[10]:= `FullSimplify[%]`

Out[10]=
$$\frac{g z \rho \left(-3 h^2 + 6 h R - 2 R^2 + \frac{4 (h - R)^4 \operatorname{Log}\left[1 - \frac{h}{R}\right]}{h (h - 2 R)} \right)}{8 \mu}$$

$Q = \int V_z(r, \theta) dA$



$$\int_R^{R-h} V_z(r) r dr$$

$$= \langle V_z \rangle$$

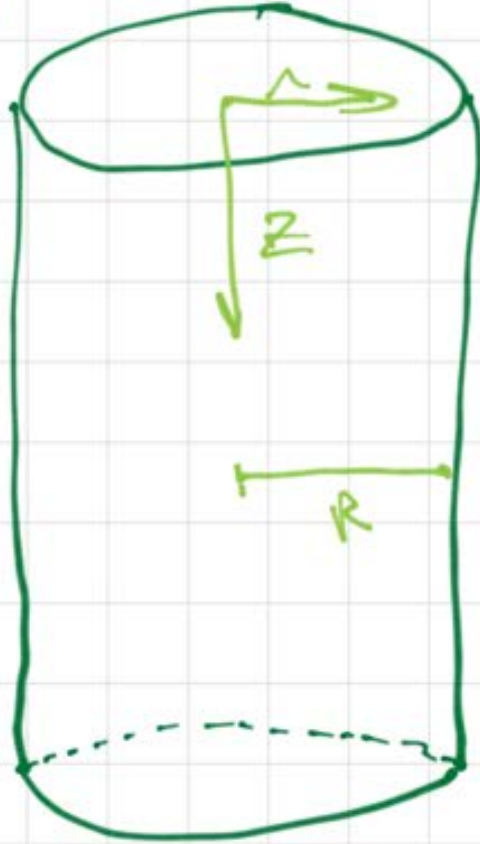
$$\int_R^{R-h} r dr$$

Suppose that we have a circular pipe oriented vertically.

z direction

$$\rho \left(\cancel{\frac{\partial v_z}{\partial t}} + \cancel{v_r \frac{\partial v_z}{\partial r}} + \cancel{\frac{v_\theta}{r} \frac{\partial v_z}{\partial \theta}} + v_z \frac{\partial v_z}{\partial z} \right) = \frac{\partial p}{\partial z} + \mu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) + \cancel{\frac{1}{r^2} \frac{\partial^2 v_z}{\partial \theta^2}} \right] - \rho g_z$$

p_1



p_2

$$0 = -\frac{\partial p}{\partial z} + \frac{\mu}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) + \rho g_z$$

BC's

$$v_z(r) = 0$$

$$\left. \frac{\partial v_z}{\partial r} \right|_{r=0} = 0$$

"SYMMETRY"

2 CASES:

1) OPEN AT BOTH ENDS $p_1 = p_2$

2) CLOSE BOTTOM: NO FLOW

LET'S DO CASE #2

$$0 = -\frac{dp}{dz} + \rho g_z$$

$$\frac{dp}{dz} = \rho g_z$$

$$\int dp = \int \rho g_z dz$$

$$p = \rho g_z z + C_0$$

$$p(0) = p_0$$

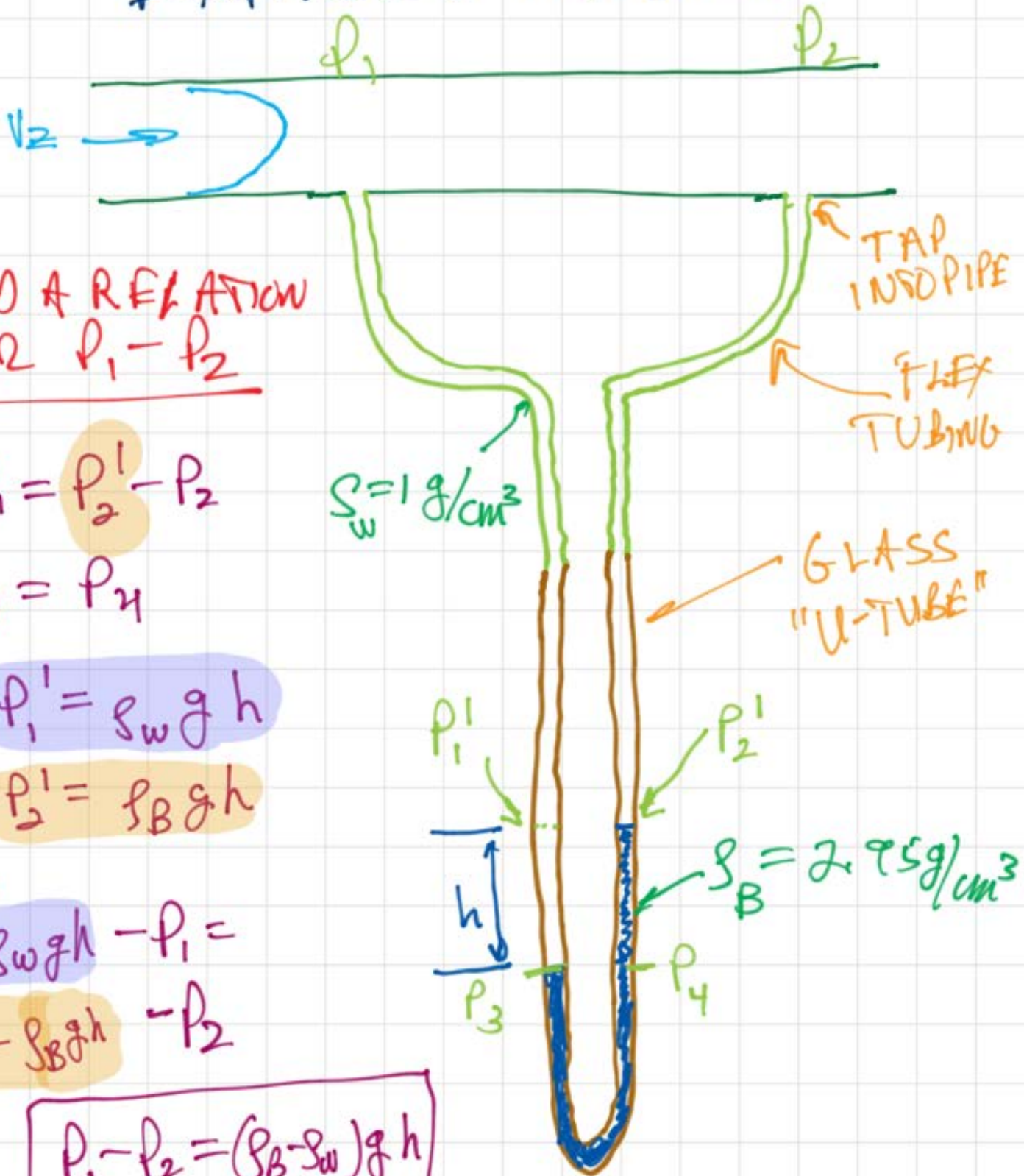
$$\therefore p = p_0 + \rho g_z z$$

p IS LINEAR IN z .

z INCREASES
DOWNWARD

BASIC EQUATION OF
HYDROSTATICS

EXAMPLE: MANOMETER



FIND A RELATION
FOR $p_1 - p_2$

$$p_1' - p_1 = p_2' - p_2$$

$$p_3 = p_4$$

$$p_3 - p_1' = \rho_w g h$$

$$p_4 - p_2' = \rho_B g h$$

SUBS:

$$p_3 - \rho_w g h - p_1 =$$

$$p_4 - \rho_B g h - p_2$$

\therefore

$$p_1 - p_2 = (\rho_B - \rho_w) g h$$

NOW CONSIDER CASE 1:
FLOW BY GRAVITY

$$0 = \frac{\mu}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) + \rho g_z$$

$$\frac{\mu}{r} \frac{d}{dr} \left(r \frac{dv_z}{dr} \right) = -\rho g_z$$

$$\frac{d}{dr} \left(r \frac{dv_z}{dr} \right) = -\frac{\rho}{\mu} g_z r$$

$$\int d \left(r \frac{dv_z}{dr} \right) = -\frac{\rho}{\mu} g_z \int r dr$$

$$r \frac{dv_z}{dr} = -\frac{\rho}{\mu} g_z \frac{r^2}{2} + C_1$$

$$dv_z = \left(-\frac{\rho}{\mu} g_z \frac{r}{2} + \frac{C_1}{r} \right) dr$$

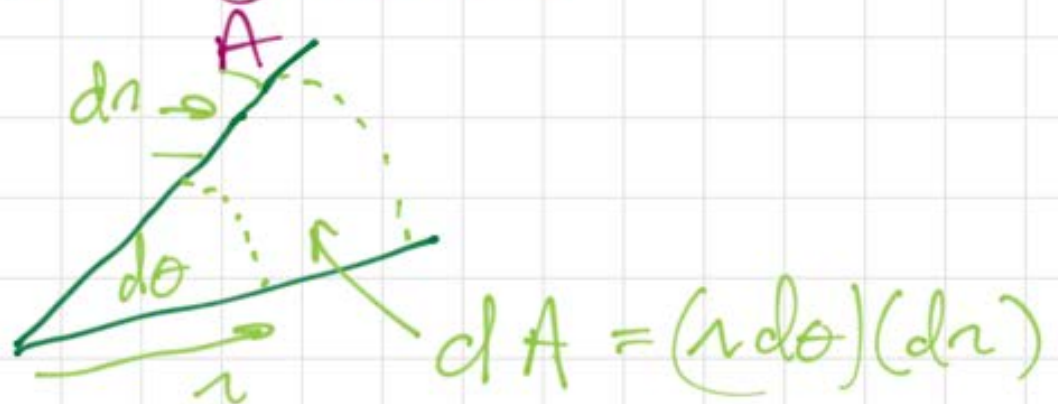
$$v_z = -\frac{\rho}{\mu} g_z \frac{r^2}{4} + C_1 \ln r + C_2$$

BUT $\zeta = 0$
FIT $v_z(r) = 0$

$$v_z(r) = \frac{\rho g z}{4\mu} (R^2 - r^2) \quad \text{PARABOLIC}$$

WE WOULD LIKE AVERAGE $\langle v_z \rangle$
NEED VOLUMETRIC FLOW

$$Q = \int v_z(r, \theta) dA$$



$$\langle v_z \rangle = \frac{\int_0^{2\pi} \int_0^R v_z(r, \theta) r dr d\theta}{\int_0^{2\pi} \int_0^R r dr d\theta}$$
$$\langle v_z \rangle = \frac{\rho g z R^2}{8\mu} = \frac{1}{2} v_{MAX}$$

$$Q = \langle V_e \rangle \pi R^2$$

$$= \frac{2\pi S g_e R^4}{8\mu}$$

VERY
SENSITIVE
TO THIS VALUE

Boundary conditions:

1. Fluid sticks to solid surfaces

$$v_x(0) = 0$$

2. Fluid sticks to another immiscible fluid

$$v_x^I(h) = v_x^{II}(h)$$

3. The shear stress is continuous across the fluid-fluid interface

$$\tau_{yx}^I(h) = \tau_{yx}^{II}(h)$$
$$\mu^I \frac{\partial v_x^I}{\partial y} \Big|_h = \mu^{II} \frac{\partial v_x^{II}}{\partial y}$$

4. For liquid flowing next to quiescent air, the stress can be considered approximately 0.

$$\tau_{yx}(h) \approx 0$$

$$\frac{\partial v_x}{\partial y}(h) \approx 0$$

"FALLING"
FILM